

**43.2** A 1000 $ft^2$  walk-in cooler using refrigerant R-134a has a capacity of 4tons. The total heat rejection from the condenser is 57MBH. At full load, what is the COP?

- A. 4
- B. 5
- C. 6
- D. 7

The **Coefficient of Performance (COP)** for a refrigerator is defined as:

$$COP_R = \frac{Q_L}{W}$$

where  $Q_L$  is the heat absorbed by the evaporator (sometimes written as  $Q_{in}$  or  $Q_{evap}$ ), and  $W$  is the work done by the compressor, (sometimes written as  $W_{in}$  or  $W_{comp}$ ).

Express  $Q_L$  in  $\frac{Btu}{hr}$ :

$$Q_L = 4 \text{ tons} \left( \frac{12000 \frac{Btu}{hr}}{\text{ton}} \right) = 48,000 \frac{Btu}{hr}$$

The heat rejected from the condenser,  $Q_H$  (sometimes  $Q_{out}$  or  $Q_{cond}$ ), is the sum of the heat absorbed by the evaporator and the energy added by the compressor. This relationship is best remembered, but could be gleaned from the Reference Handbook section **Basic Cycles**, under **Thermodynamic Cycles**.

Rearrange to isolate  $W$  and solve, converting  $MBH$  to  $\frac{Btu}{hr}$  as required:

$$Q_H = Q_L + W \rightarrow W = Q_H - Q_L$$

$$W = 57MBH \left( \frac{10,000 \frac{Btu}{hr}}{MBH} \right) - 48,000 \frac{Btu}{hr} = 9000 \frac{Btu}{hr}$$

Solve for the  $COP_R$ :

$$COP_R = \frac{Q_L}{W} = \frac{48,000 \frac{Btu}{hr}}{9000 \frac{Btu}{hr}} = 5.3$$

**Answer B**

**43.3** A 1000cfm fan coil unit installed for dehumidification maintains an apparatus dew point of 40°F and has a coil bypass factor of 10%. The entering air is 80°F dry bulb and 50% relative humidity. What quantity of total cooling is provided?

- A. 3.2tons
- B. 3.6tons
- C. 5.4tons
- D. 6.0tons

The bypass factor is the portion of the air flow that gets by the coil without being cooled. The bypass factor is the complement of the coil efficiency. When the bypass factor is 0, the coil is 100% efficient. In this case,

$$BF = 1 - \eta_{coil} = .1$$

$$\eta_{coil} = .9$$

The coil efficiency can be expressed as a ratio of the reduction in dry bulb temperature actually achieved as compared with what could possibly be achieved by the supply air temperature converging with the apparatus dew point i.e. the physical temperature of the coil. Let  $T_1$  represent the entering air condition,  $T_2$  represent the supply air condition, and  $ADP$  represent the coil condition. Write an expression for the coil efficiency:

$$\eta_{coil} = \frac{T_1 - T_2}{T_1 - T_{ADP}} = .9$$

Substituting known entering conditions and coil conditions, solve for the supply air temperature leaving the coil:

$$\frac{80 - T_2}{80 - 40} = .9 \rightarrow T_2 = 44^\circ F$$

If the problem were asking for the quantity of sensible cooling provided, we would now have enough information since only the dry bulb temperatures would be needed. However, in order to find the total cooling provided, the supply air condition must be fully defined.

Use the psychrometric chart to look up the enthalpy and humidity ratio for the entering air:

$$T_1 = 80^\circ F$$

$$\phi_1 = 50\%$$

$$h_1 = 31.2 \frac{Btu}{lb}$$

$$\omega_1 = 76.4 \frac{gr}{lb}$$