

$$\dot{Q}_{total} = 3750 \frac{Btu}{hr} + 10,000 \frac{Btu}{hr} + 2880 \frac{Btu}{hr} + 6480 \frac{Btu}{hr} = 23,110 \frac{Btu}{hr}$$

**Answer D**

**43.9** In an effort to allow for increased ceiling heights, a design engineer specifies a 3:1 aspect ratio for a duct that delivers  $500\text{cfm}$ . The velocity is to be kept under  $400\text{fpm}$ . What is the dimension of the long side of the duct?

- A.  $13\text{in}$
- B.  $14\text{in}$
- C.  $23\text{in}$
- D.  $24\text{in}$

Solve for the minimum area required to keep the velocity under the maximum:

$$Q = VA$$

$$A = \frac{Q}{V} = \frac{500 \frac{ft^3}{min}}{400 \frac{ft}{min}} = 1.25 ft^2 \left( \frac{12in}{1ft} \right)^2 = 180in^2$$

Let the short side of the duct have length  $x$  and the long side have a length of  $3x$  to achieve a 3:1 aspect ratio. Write an expression for the area and solve for the length of the short side,  $x$ .

$$3x^2 = 180in^2$$

$$x = 7.74in$$

The duct dimensions must be integers. Since  $7 < 7.74 < 8$ , the short side may be  $7\text{in}$  or  $8\text{in}$  in length. Generate several possible sets of dimensions for consideration:

$$7 \times 23 \rightarrow A = 161in^2 < 180in^2 \text{ (insufficient)}$$

$$7 \times 24 \rightarrow A = 168in^2 < 180in^2 \text{ (insufficient)}$$

$$8 \times 23 \rightarrow \text{(not 3 : 1 ratio)}$$

$$8 \times 24 \rightarrow A = 192in^2 > 180in^2, 3 : 1 \text{ ratio OK, } V < 400\text{fpm}$$

The long side is  $24\text{in}$ .

**Answer D**

**43.10** The pressure loss in a duct is  $1.2\text{in wg}$  when the flow rate is  $2500\text{cfm}$ . If the fan slows down to  $2000\text{cfm}$ , what is the new pressure under reduced speed conditions?

- A.  $0.8\text{in wg}$
- B.  $1.0\text{in wg}$
- C.  $1.5\text{in wg}$
- D.  $1.9\text{in wg}$

Look up **Fan Affinity Laws** in the Reference Handbook and see the table called **Fan Laws**. Choose Law #3b:

$$P_1 = P_2 \times \left(\frac{D_2}{D_1}\right)^4 \left(\frac{Q_1}{Q_2}\right)^2 \left(\frac{\rho_1}{\rho_2}\right)$$

There is no indication that the fan diameter is being changed. Therefore assume  $D_1 = D_2$ . The density of the air under the new conditions may also be assumed to be unchanged. Therefore assume  $\rho_1 = \rho_2$ . Simplify:

$$P_1 = P_2 \times \left(\frac{Q_1}{Q_2}\right)^2$$

Note: It may be advisable to memorize some common uses of the fan laws where only one parameter is being adjusted. It is common to recall that pressure is a function of the volume flow rate squared.

For convenience, swap the subscripts and make a quick table to ensure proper handling of knowns and unknowns. Let condition 1 be the existing fan and condition 2 be for reduced speed.

$$P_2 = P_1 \times \left(\frac{Q_2}{Q_1}\right)^2$$

| Parameter            | Condition 1 | Condition 2 |
|----------------------|-------------|-------------|
| $Q_{[\text{cfm}]}$   | 2500        | 2000        |
| $P_{[\text{in wg}]}$ | 1.2         | $P_2$       |

Solve for  $P_2$  :

$$P_2 = 1.2\text{in wg} \times \left(\frac{2000\text{cfm}}{2500\text{cfm}}\right)^2 = .77\text{in wg}$$

Note there is a nonlinear reduction in pressure. This should be consistent with intuition as pressure changes with the square of the speed (and volume flow rate).

**Answer A**