

atmospheric conditions. Use the **Properties of Saturated Water and Steam** (Pressure) table to obtain the value of h_{fg} at atmospheric pressure.

$$P = 14.7 \text{ psia}$$

$$h_{fg} \approx 970 \frac{\text{Btu}}{\text{lb}}$$

Hint: When making approximations of latent load due to moisture, consider assuming $h_{fg} \approx 1000 \frac{\text{Btu}}{\text{lb}}$ to save time.

Calculate the latent load.

$$\dot{Q}_L = \dot{m}\Delta h = \dot{m}\Delta h_{fg} = \left(12 \frac{\text{lb}}{\text{hr}}\right) \left(970 \frac{\text{Btu}}{\text{lb}}\right) = 11,640 \frac{\text{Btu}}{\text{hr}}$$

Solve for SHR .

$$SHR = \frac{\dot{Q}_S}{\dot{Q}_S + \dot{Q}_L} = \frac{34,120 \frac{\text{Btu}}{\text{hr}}}{34,120 \frac{\text{Btu}}{\text{hr}} + 11,640 \frac{\text{Btu}}{\text{hr}}} = 0.746$$

Answer C

47.15 A 10 lb_m mass hangs from a spring and damper assembly. The spring has a spring constant of $100 \frac{\text{lb}_f}{\text{in}}$. The damping ratio is 0.6. What is the damped frequency of the system?

- A. 6 Hz
- B. 8 Hz
- C. 10 Hz
- D. 12 Hz

Find the natural frequency of the system as though no damping was present. The **undamped natural circular frequency** is given by the equation below.

$$\omega_n = \sqrt{\frac{kg_c}{m}} = \sqrt{\frac{\left(100 \frac{\text{lb}_f}{\text{in}}\right) \left(12 \frac{\text{in}}{\text{ft}}\right) \left(32.2 \frac{\text{lb}_m \cdot \text{ft}}{\text{lb}_f \cdot \text{s}^2}\right)}{10 \text{ lb}_m}} = 62.2 \frac{\text{rad}}{\text{s}}$$

Find the **Damped Natural Frequency** which accounts for the damping ratio.

$$\omega_d = \omega_n \sqrt{1 - \zeta^2} = \left(62.2 \frac{\text{rad}}{\text{s}}\right) \sqrt{1 - (0.6)^2} = 49.7 \frac{\text{rad}}{\text{s}}$$

Since the answer choices are in Hz , find the corresponding damped *linear* frequency. Note the final units of “cycles per second” is the same as Hz .

$$f_d = \frac{\omega_d}{2\pi} = \frac{49.7 \frac{\text{rad}}{\text{s}}}{2\pi \frac{\text{rad}}{\text{cycle}}} = 7.9 \text{ Hz}$$

Answer B

47.16 A Carnot heat pump operates during winter when the outside air temperature is $20^{\circ}F$ and inside temperature is $68^{\circ}F$. During summer, the unit runs in reverse to provide cooling when the outside air temperature is $100^{\circ}F$ and inside temperature is $72^{\circ}F$. What is the coefficient of performance during summer operation?

- A. 3
- B. 4
- C. 17
- D. 19

Recall that a **Carnot** heat pump has the maximum theoretical **Coefficient of Performance** based on the temperatures of the reservoirs heat is being removed from and rejected to. In this case, winter operation may be ignored since the question is asking about the **COP** for summer operation only.

The COP for refrigeration for a Carnot cycle is given by:

$$COP_c = \frac{T_L}{T_H - T_L}$$

Temperature units must be absolute (Rankine). The denominator is a temperature differential and therefore may be left in Fahrenheit.

$$COP_c = \frac{72^{\circ}F + 460^{\circ}}{100^{\circ}F - 72^{\circ}F} = 19$$

Answer D

47.17 A refrigeration cycle using R-123 operates between 30psia and 125psia with no subcooling and isenthalpic expansion. What is the quality of the mixture entering the evaporator?

- A. 0.20
- B. 0.26
- C. 0.32
- D. 0.38

Look up the **R-123** properties table and **Pressure Versus Enthalpy Curves for Refrigerant 123**. Consider the high pressure side of the refrigeration cycle after the refrigerant passes through the condenser, it is expected to be a saturated liquid at 125psia . Let this be considered State 3. Use either the table or the chart to determine the enthalpy of saturated liquid, h_f . The table lends itself to slightly higher precision but may take a bit more time. Formal interpolation is not necessary. Beware of the logarithmic scale on the vertical axis if using the P-H chart. Since the expansion process in a typical refrigeration cycle is isenthalpic, the enthalpy after expansion is the same as the enthalpy prior to expansion. Let the low pressure condition after expansion be considered State 4.